



## Federation of European Heating, Ventilation and Air Conditioning Associations

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Federation of European Heating, Ventilation and Air-conditioning Associations

# CFD in Ventilation Design

by

Peter V. Nielsen,

Aalborg University

# The Audience

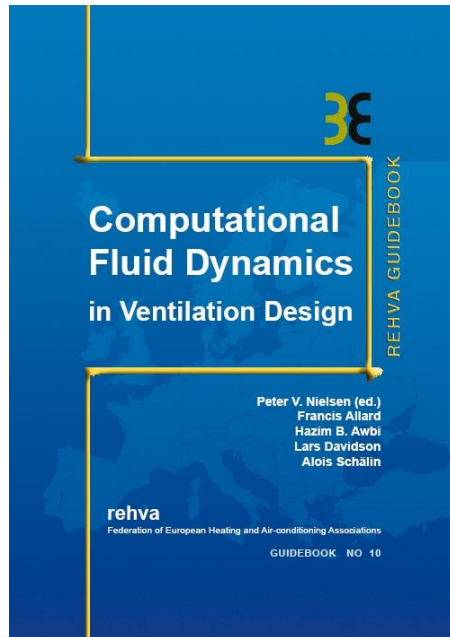
The user of the design book is mainly considered to be a consulting engineer who has to:

- order a CFD prediction
- consider and work with a CFD prediction
- discuss CFD and the CFD quality with a supplier of a CFD prediction

In principle the book is not written for engineers who are making CFD predictions already

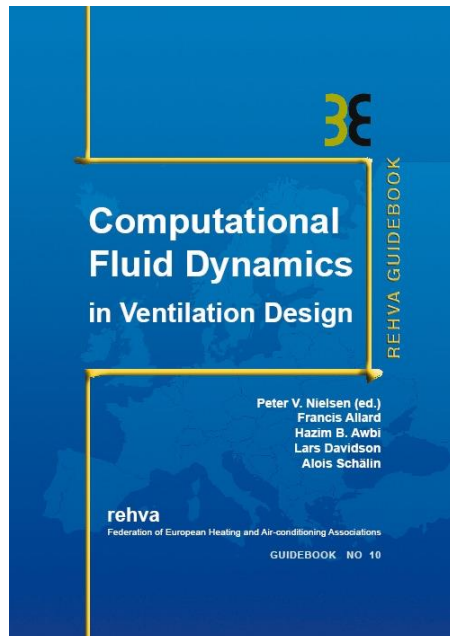
# CFD in Ventilation Design

**Authors:** Francis Allard, Hazim B. Awbi  
Lars Davidson, Alois Schälin and Peter V. Nielsen



Computational fluid dynamics in a nutshell  
Symbols and glossary  
Mathematical background  
Turbulence models  
Numerical methods  
Boundary conditions  
Quality control  
CFD combined with other prediction models  
Application of CFD codes in building design  
Case studies  
Benchmark tests

# CFD in Ventilation Design



## **Computational fluid dynamics in a nutshell**

Symbols and glossary

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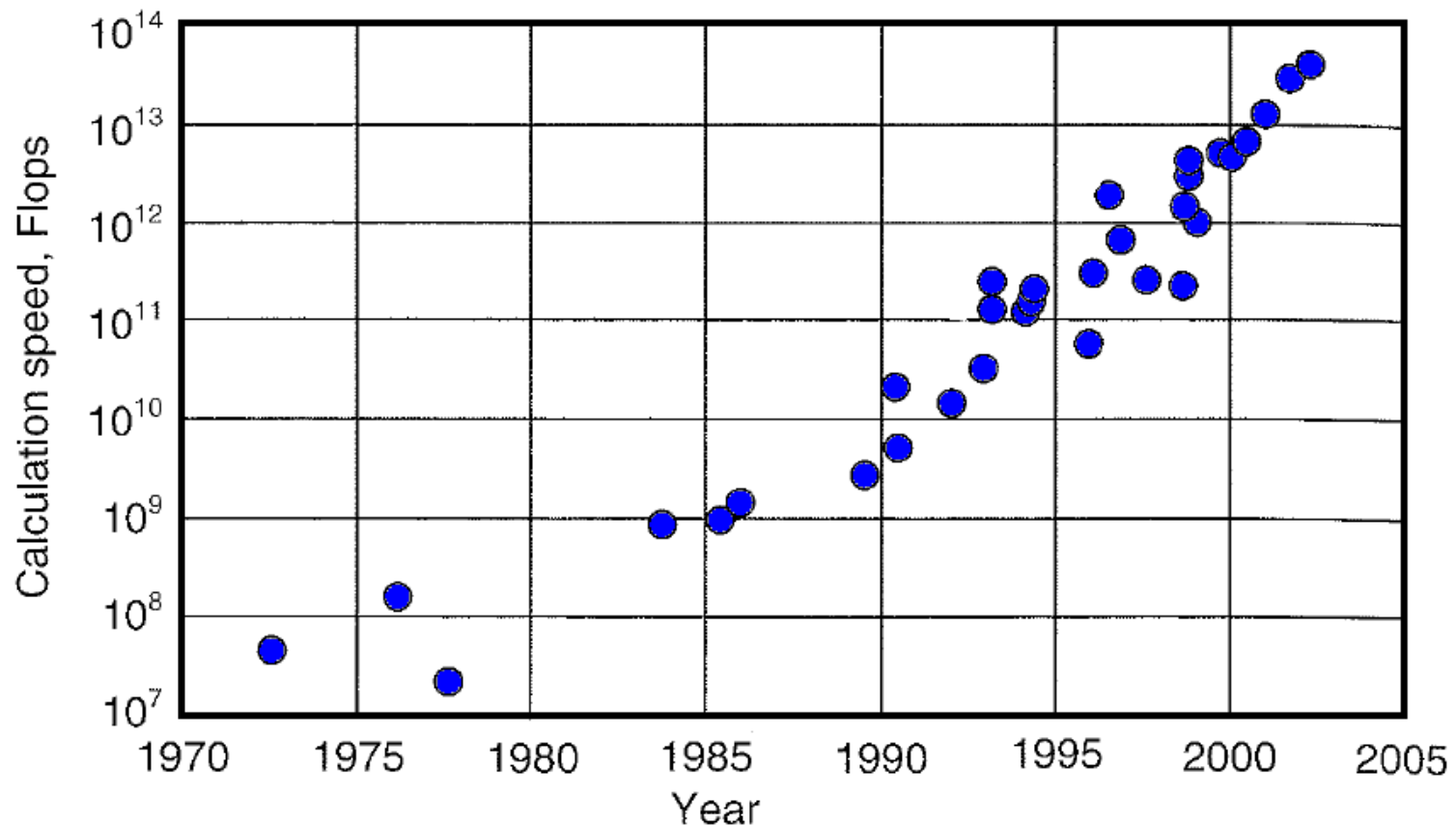
CFD combined with other prediction models

Application of CFD codes in building design

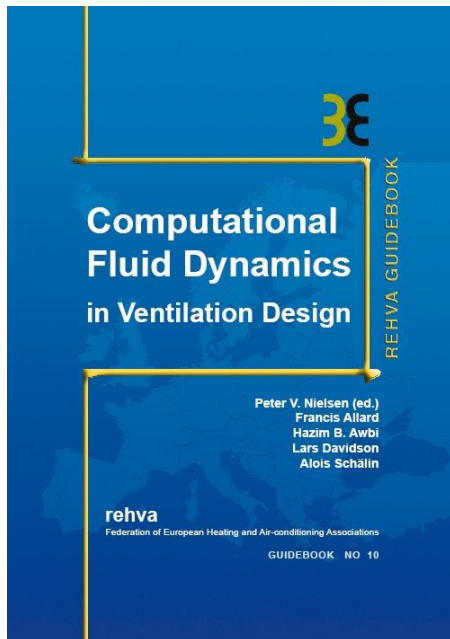
Case studies

Benchmark tests

# Development in Computer Cost



# CFD in Ventilation Design



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# Mathematical Background

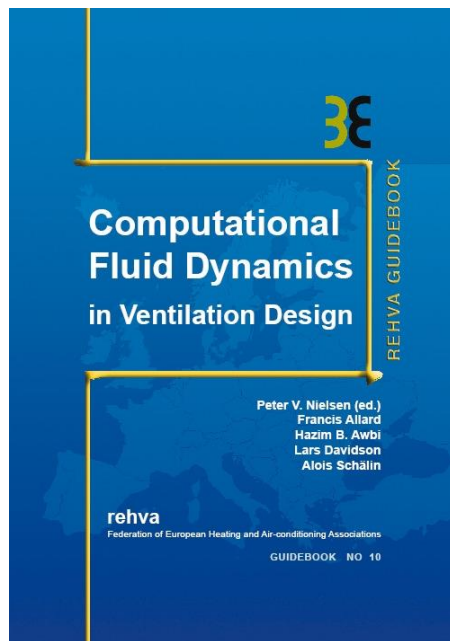
From the general description

$$\frac{\partial(\rho\Phi)}{\partial t} + \text{div}(\rho\vec{V}\Phi) = \text{div}(\Gamma_{\Phi} \overrightarrow{\text{grad}}\Phi) + S_{\Phi}$$

to a two-dimensional time dependent transport equation

$$\frac{\partial c}{\partial t} + u \frac{\partial c}{\partial x} + v \frac{\partial c}{\partial y} = D \left( \frac{\partial^2 c}{\partial x^2} + \frac{\partial^2 c}{\partial y^2} \right)$$

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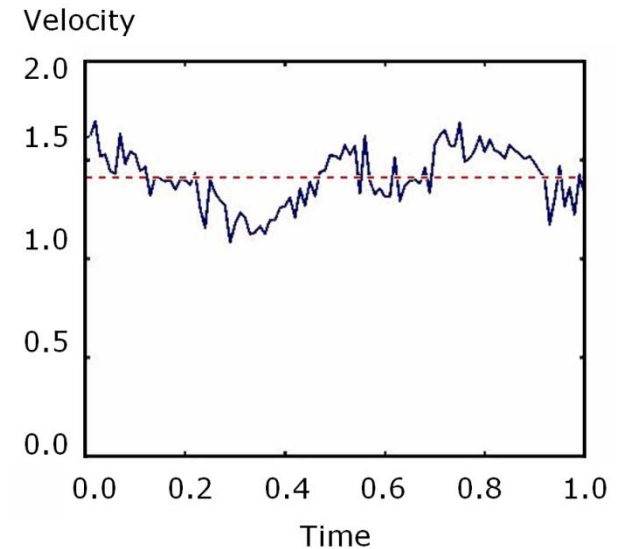
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# Turbulence Models

Laminar flow/turbulent flow  
2D steady state

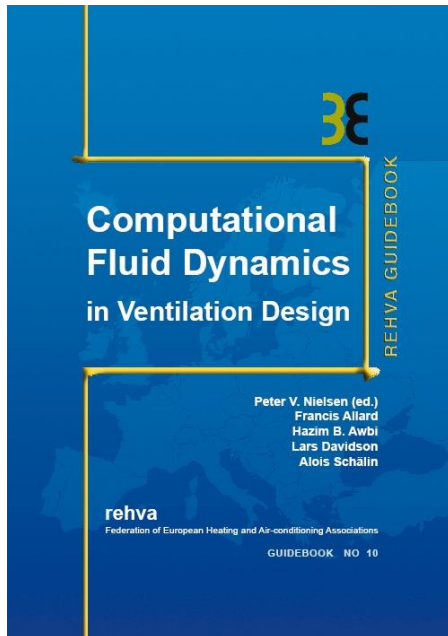
$$\rho u \frac{\partial c}{\partial x} + \rho v \frac{\partial c}{\partial y} = \Gamma_c \frac{\partial^2 c}{\partial y^2} + S_c$$



A discussion of different turbulence models as e.g.  
the  $k$ - $\varepsilon$  model, the  $k$ - $\omega$  model, the  $SST$  model  
and the Reynolds Stress model

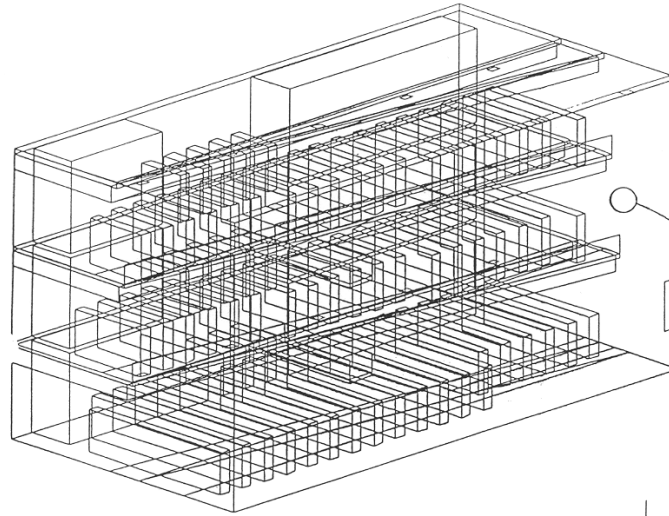
The Large Eddy Simulation is also discussed

# CFD in Ventilation Design

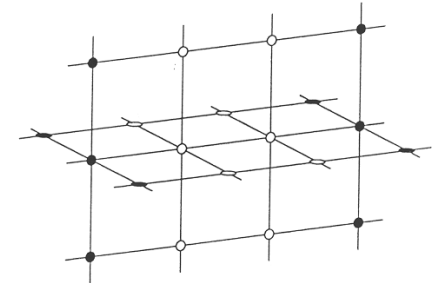


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# One-Dimensional Case

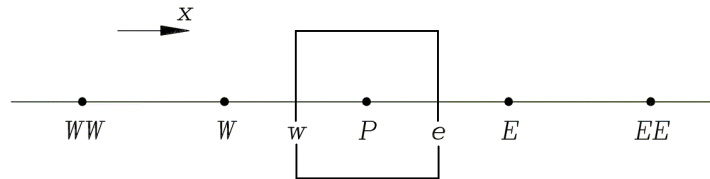


The case can be considered as a small part of a flow, which in certain areas is one-dimensional, parallel with grid lines and steady.



$$\rho u \frac{dc}{dx} = \Gamma_c \frac{d^2c}{dx^2} + S_c$$

# One-Dimensional Discretization Equation



Control volume formulation:

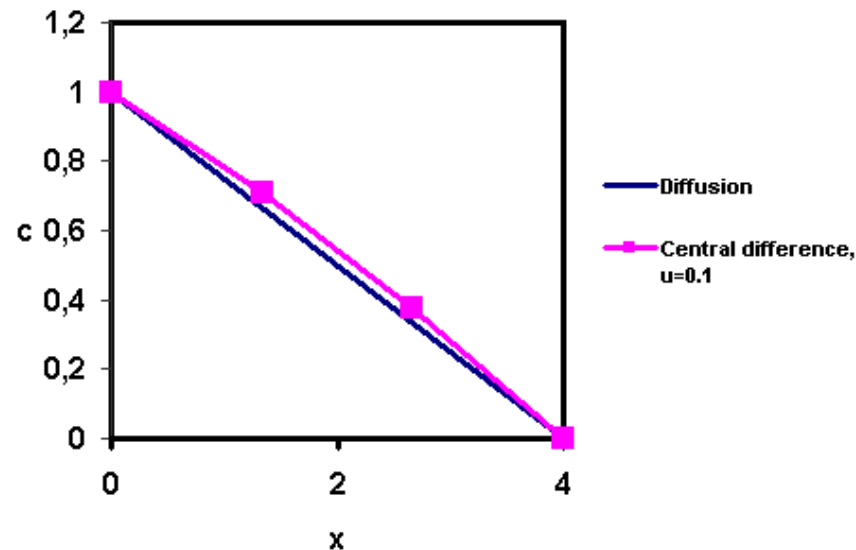
$$\rho(u_e c_e - u_w c_w) = \Gamma_c \left[ \left( \frac{dc}{dx} \right)_e - \left( \frac{dc}{dx} \right)_w \right] + S_P \Delta x$$

It is necessary to replace values at the cell surfaces  $e$  and  $w$  with values from the grid points  $WW$ ,  $W$ ,  $P$ ,  $E$  and  $EE$  to have a final version of the discretization equation.

# Different Discretization Equations, 1

The flow is studied in a case where the length  $x$  is equal to 4. The boundary values  $c_0$  and  $c_3$  are equal to 1.0 and 0.0.

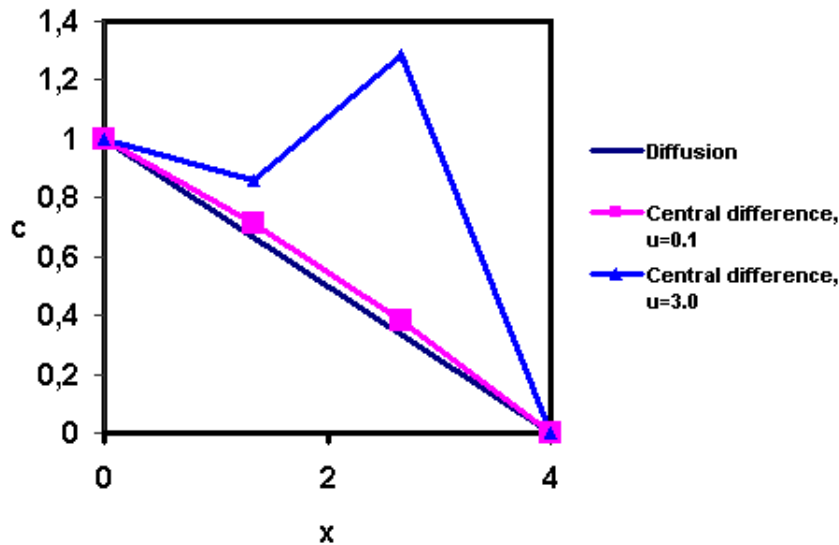
Central differences is e.g.  $c_e = (c_p + c_E)/2$  and  $u = 0.1$



# Different Discretization Equations, 2

Central differences e.g.  $c_e = (c_P + c_E)/2$  and  $u = 3.0$

Wiggly for  $Pe = \frac{\rho \Delta x u}{\Gamma_c}$  larger than 2





# History and Numerical Schemes, 1

## **The sixties**

The central difference scheme becomes unstable (wiggly) when the Peclet number is large. The cure is to decrease the grid size.

## **The seventies**

Upwind difference opened the way for infinitely high Reynolds numbers, but false diffusion could in many cases be larger than diffusion of physical kind.

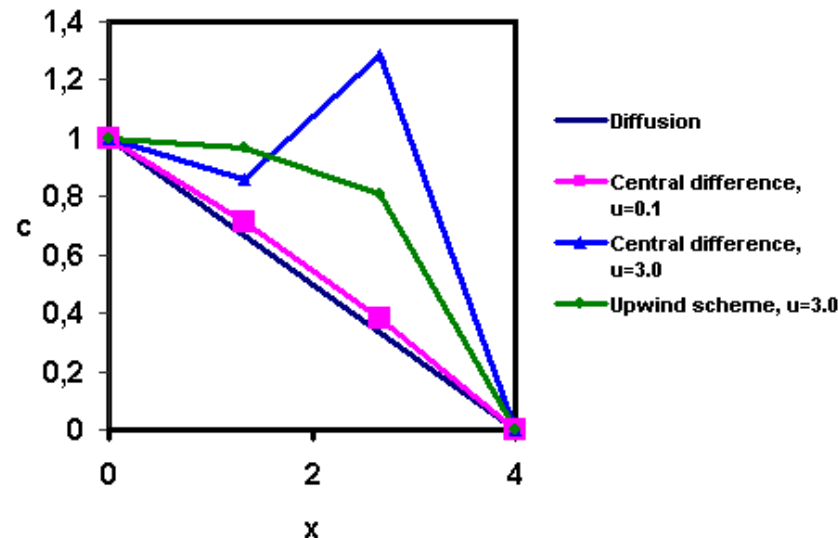
## **The eighties and the nineties**

Second order schemes decreased the effect of false diffusion.

# Different Discretization Equations, 3

The flow is studied in a case where the length  $x$  is equal to 4. The boundary values  $c_0$  and  $c_3$  are equal to 1.0 and 0.0.

Upwind scheme e.g.  $c_e = c_p$  and  $u = 3.0$



# History and Numerical Schemes, 2

## **The sixties**

The central difference scheme becomes unstable (wiggly) when the Peclet number is large. The cure is to decrease the grid size.

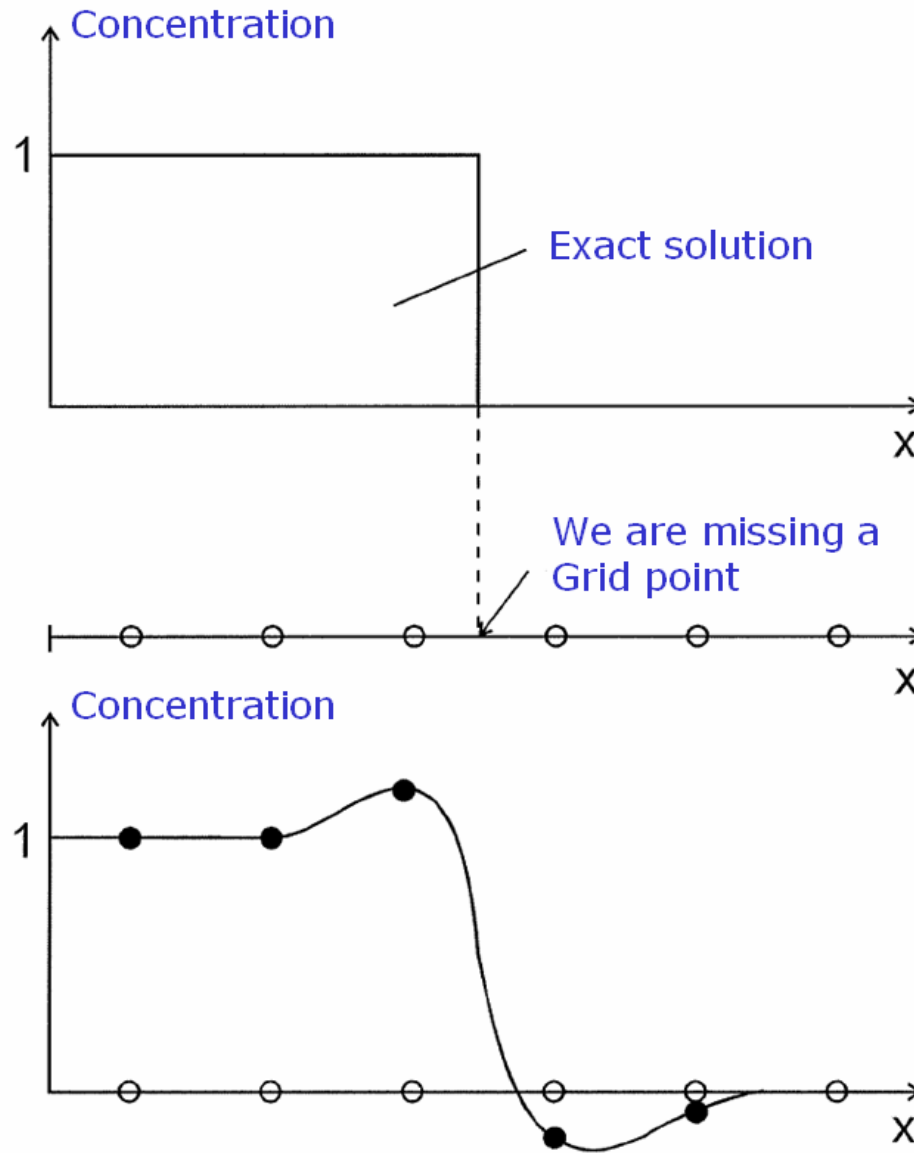
## **The seventies**

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## **The eighties and the nineties**

Second order schemes decreased the effect of false diffusion.

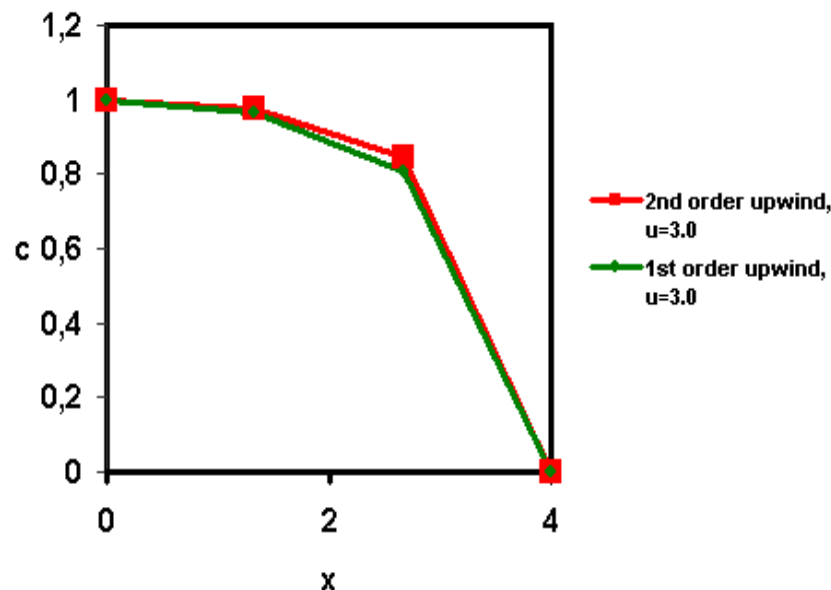
# False Diffusion



# Higher Order Schemes

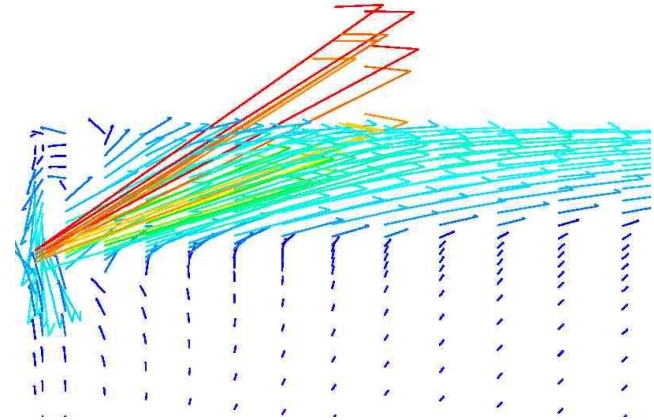
Second order upwind scheme is an example of a new scheme developed in the middle of the seventies.

The one-dimensional case with the velocity  $u = 3.0$  and 1st and 2nd order upwind scheme

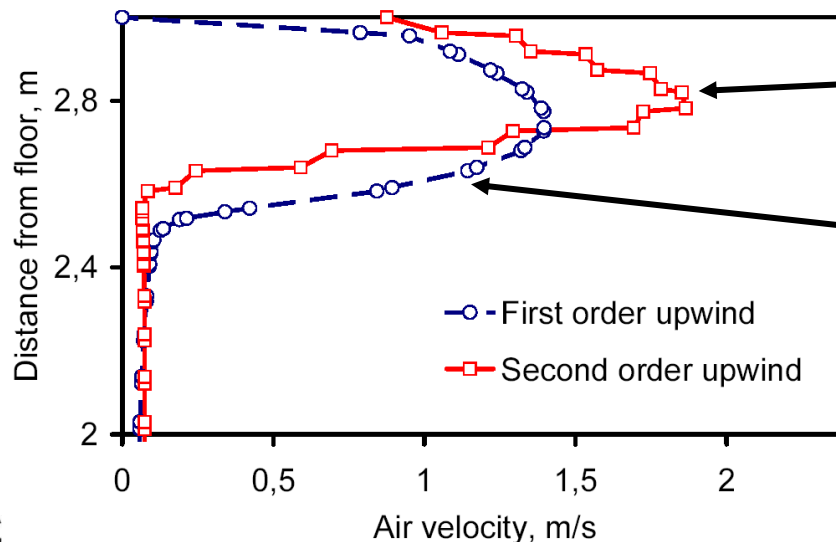


# False Diffusion and Order of the Schemes

Flow from an opening, which is inclined at  $\sim 30$  deg. to the mesh.  
Three-dimensional flow.



Profile at the upper surface at a distance of 1 m from the opening.



Dispersive error

Diffusive error

# History and Numerical Schemes, 3

## **The sixties**

The central difference scheme becomes unstable (wiggly) when the Peclet number is large. The cure is to decrease the grid size.

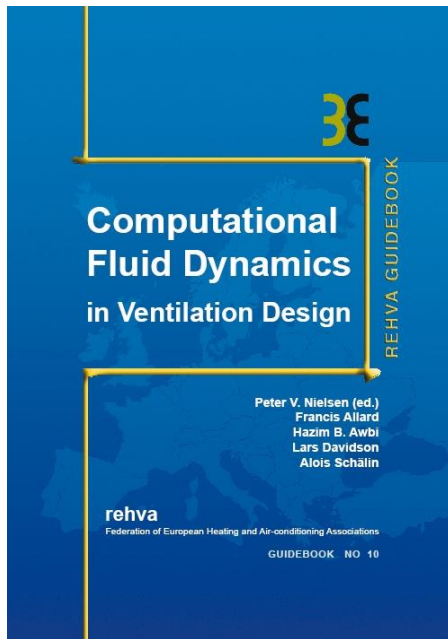
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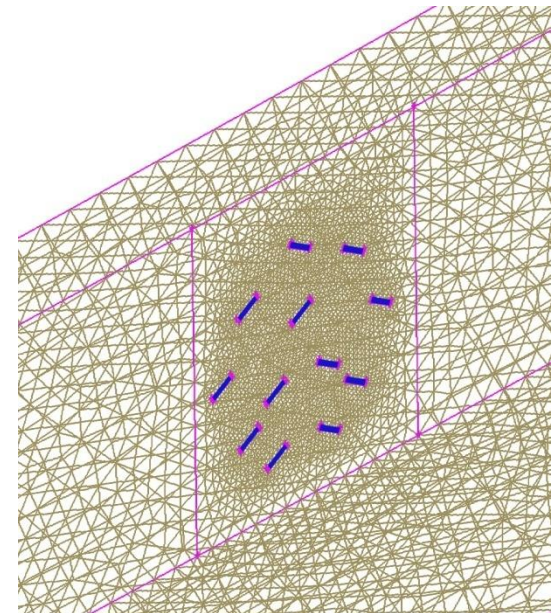
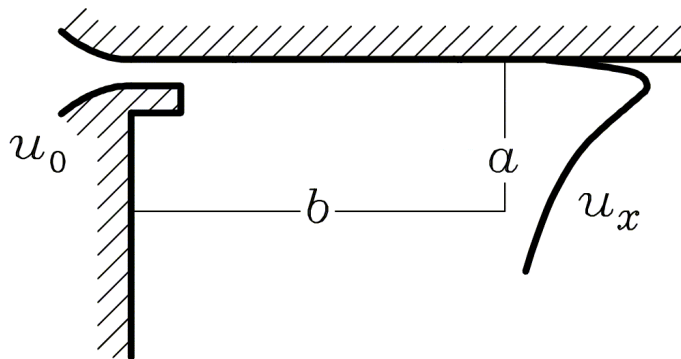
Benchmark tests



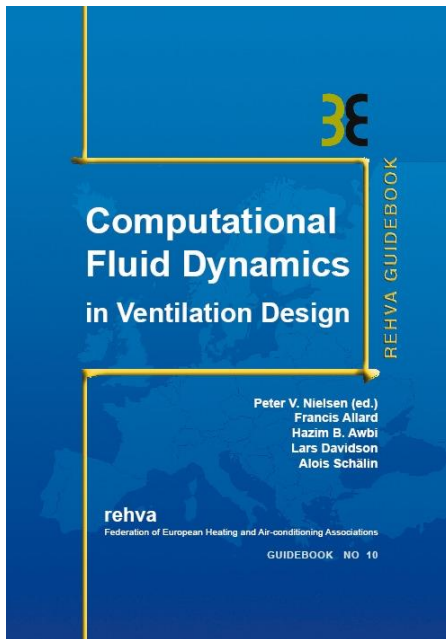
# Boundary Conditions

- Wall boundary
- Free boundary
- Plane of symmetry
- Air supply opening
- Air exit opening
- Obstacle boundary

Air supply opening



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# Quality Control

Quality control consists of these major steps:

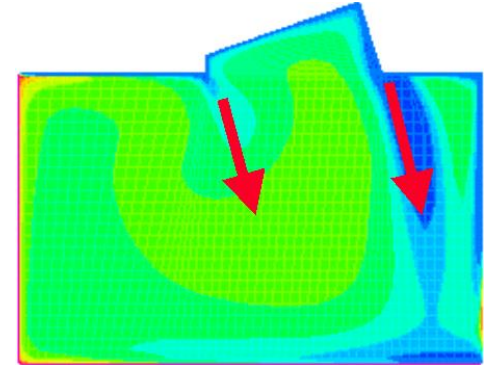
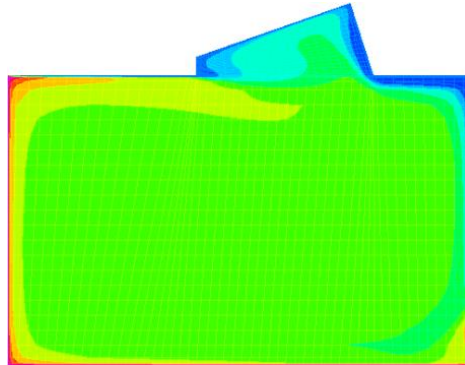
- recognize possible sources of errors
- check for these errors in your own simulations
- estimate the accuracy of the simulations
- improve the simulations, if possible

Main items in this chapter are:

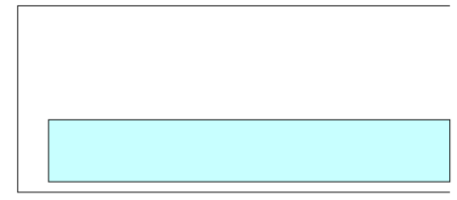
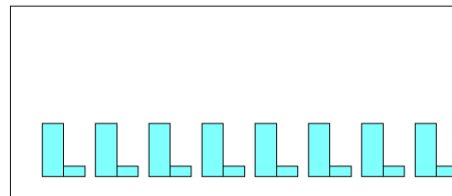
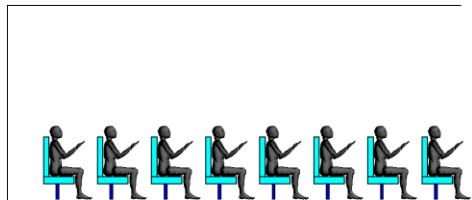
- Steps in a CFD simulation
- Sources of errors and uncertainties
- How to ensure high quality predictions (recommendations)
- Questions to ask the CFD engineer as regards the work reported
- Additional advice and remarks
- A short check list

# Quality Control, Sources of Errors and Uncertainties

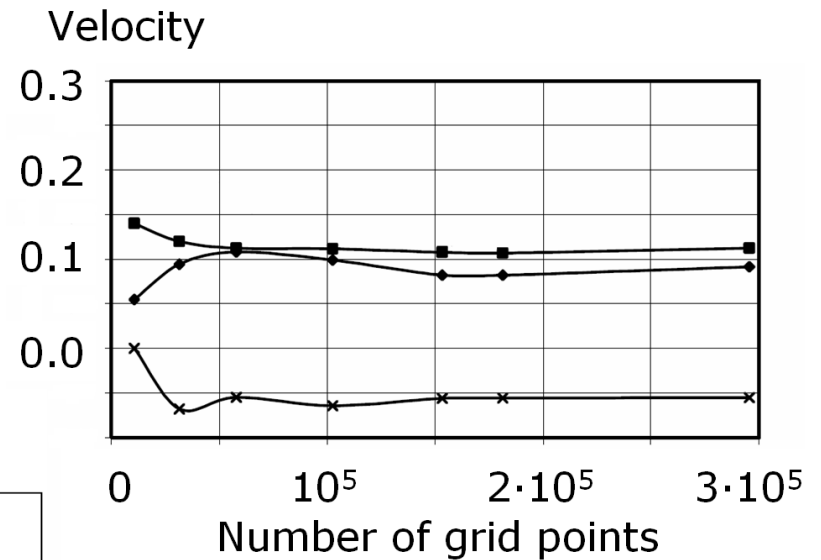
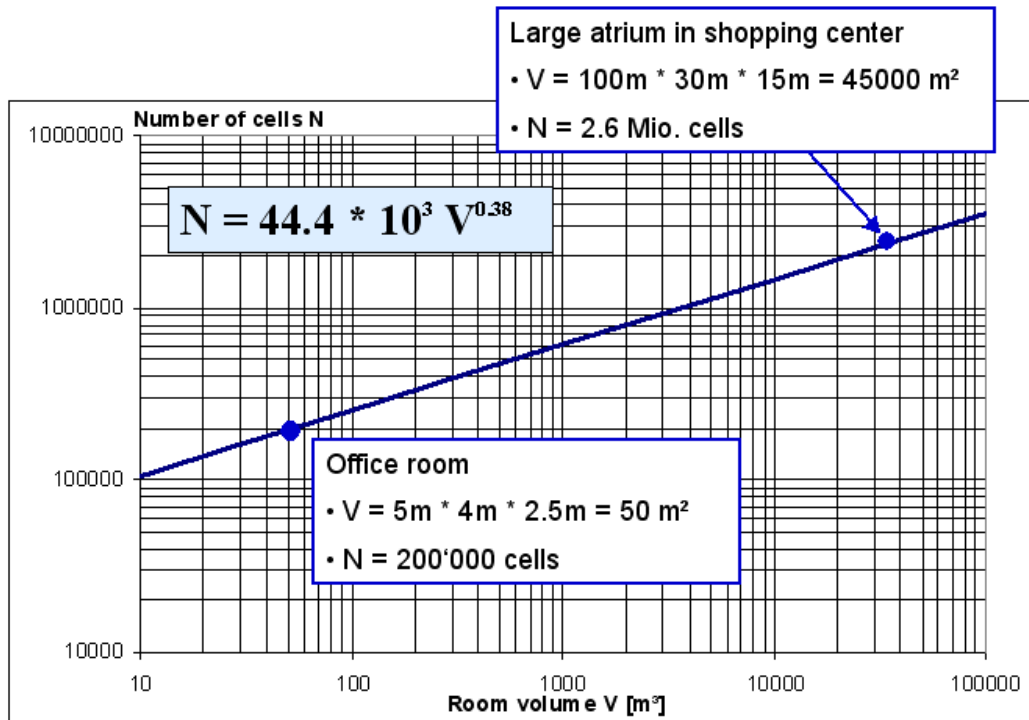
Some examples: 2D treatment instead of 3D



Simplification, modelling level

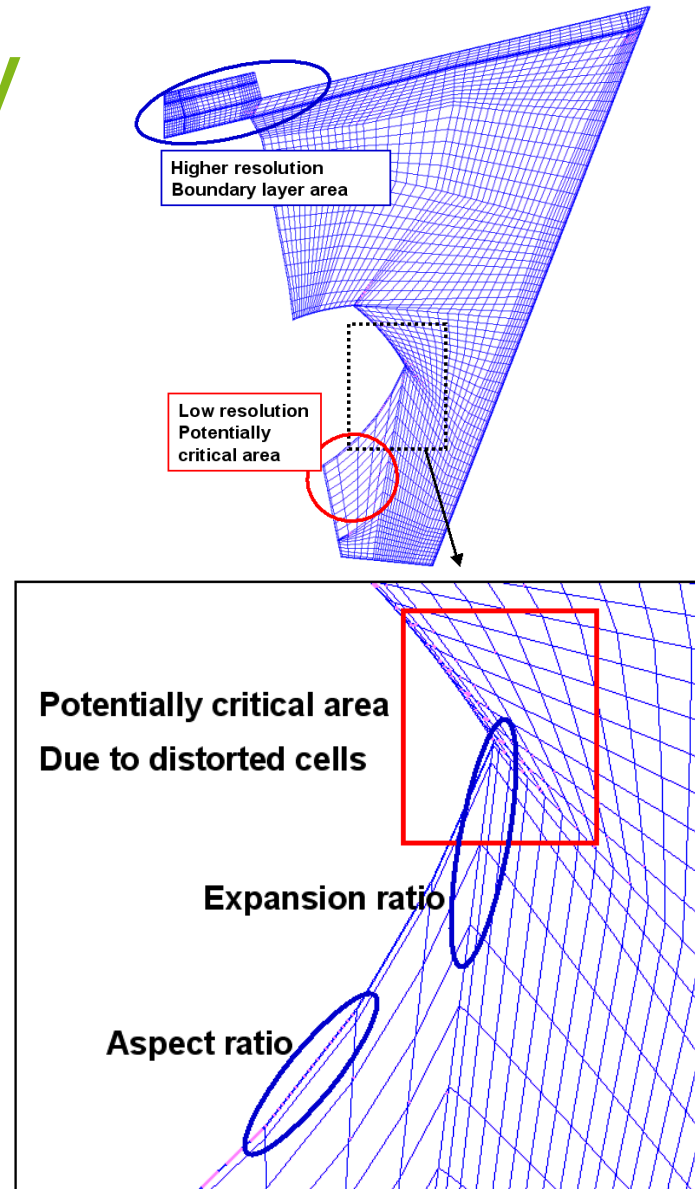


# Cell Quality



Monitoring velocities  
versus number of cells  
in the prediction

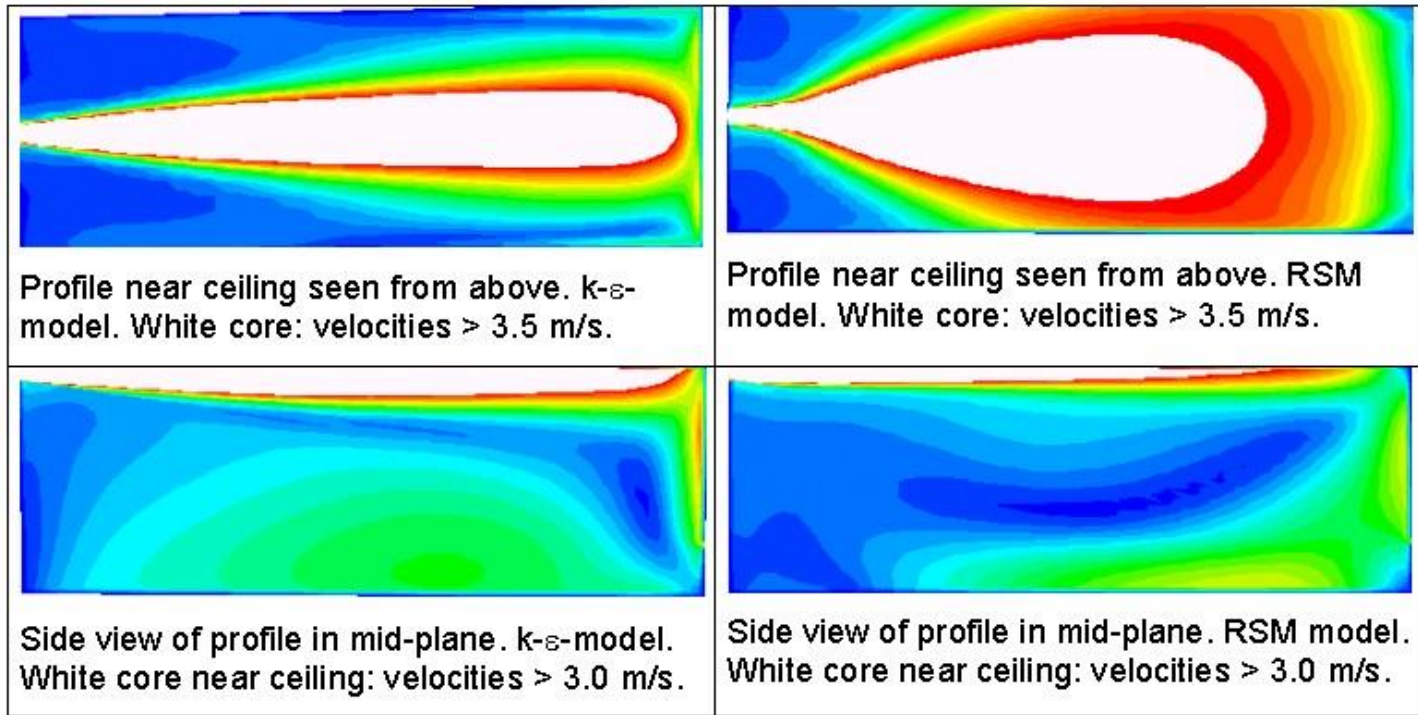
# Cell Quality





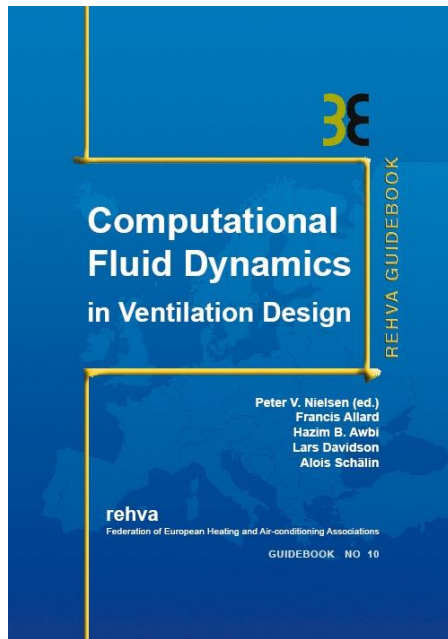
# Turbulence Model

3D wall jet in a room simulated by a *k-epsilon Model* and a *Reynolds Stress Model*.



Schälin and Nielsen

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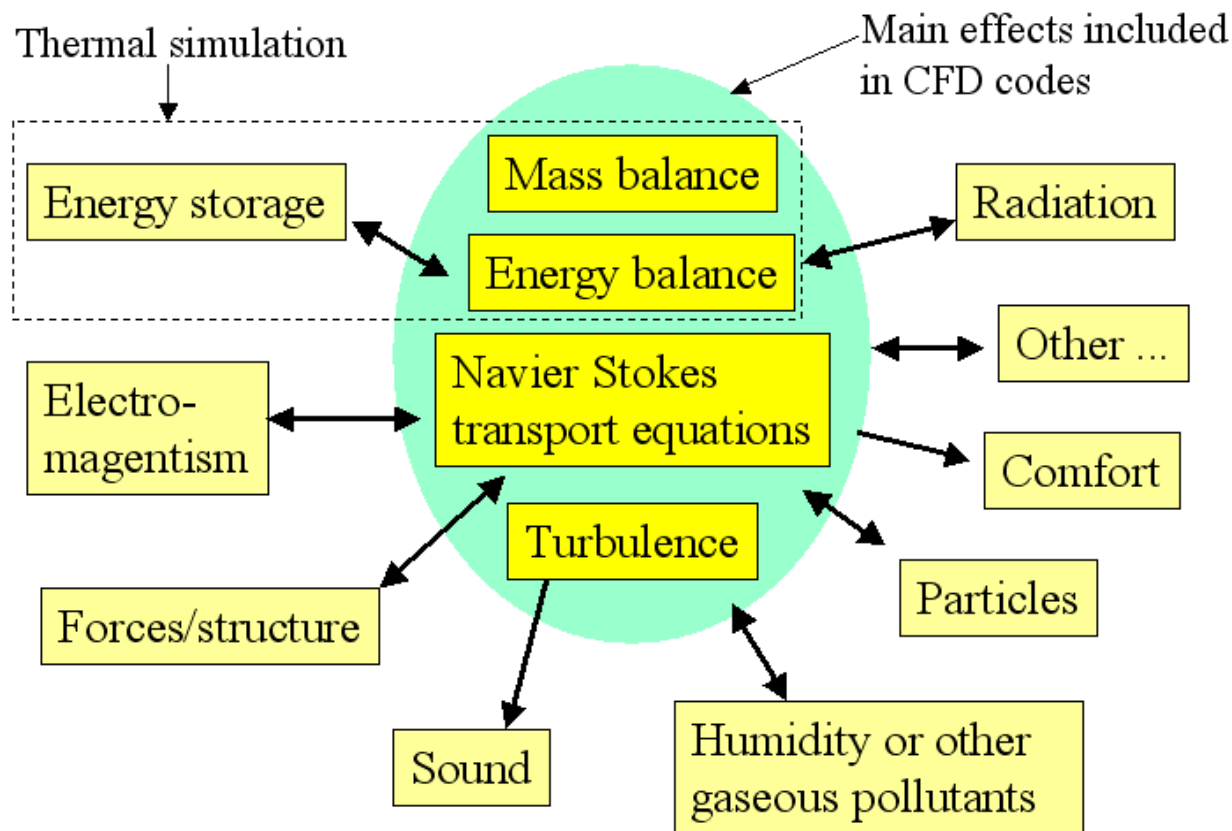
Application of CFD codes in building design

Case studies

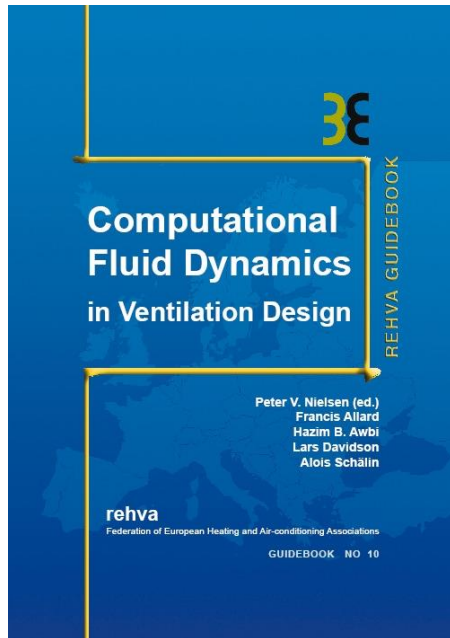
Benchmark tests



# CFD Combined with Other Prediction Models



# CFD in Ventilation Design



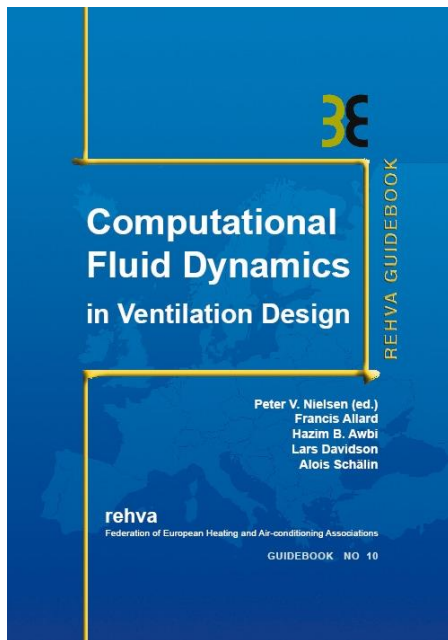
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# Application of CFD Codes in Building Design

The applications of CFD in buildings may be grouped under the following headings:

- Prediction of air jet diffusion
- Analysis of Room air movement
- Prediction of contaminant dispersal
- Modelling emission from materials and equipment in buildings
- Indoor air quality prediction
- Thermal comfort assessment
- Mean age of air and ventilation effectiveness predictions
- Prediction of fire and smoke spread
- Wind flow around buildings

# CFD in Ventilation Design

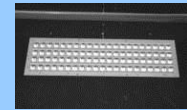


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# Case Studies on Different Air Distribution Systems

Five air distribution systems are compared with each other. They are all installed in the same room, and they all handle the same situation and the same load.

**Mixing ventilation with end wall mounted diffuser.**



**Vertical ventilation with a textile terminal..**



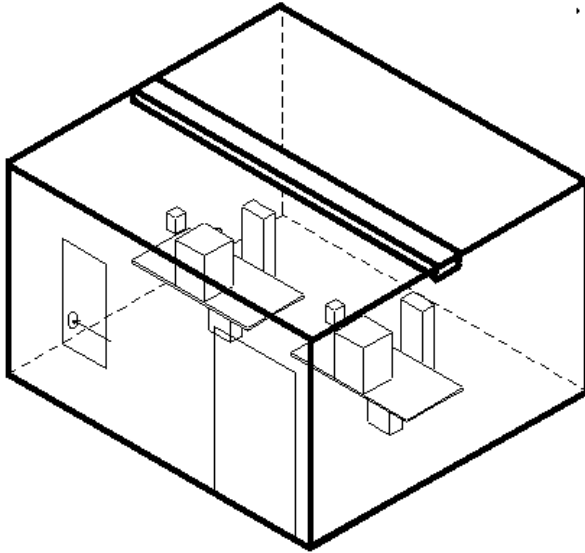
**Displacement ventilation. End wall mounted low velocity diffuser.**



**Mixing ventilation generated by a ceiling mounted radial diffuser.**



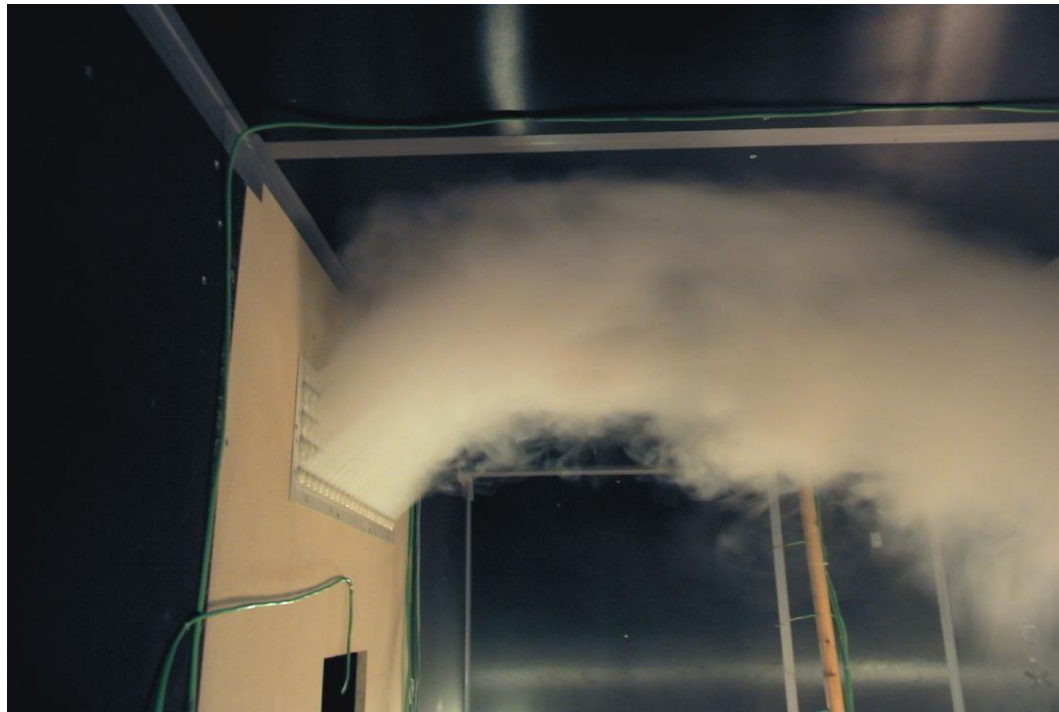
# The Test Room



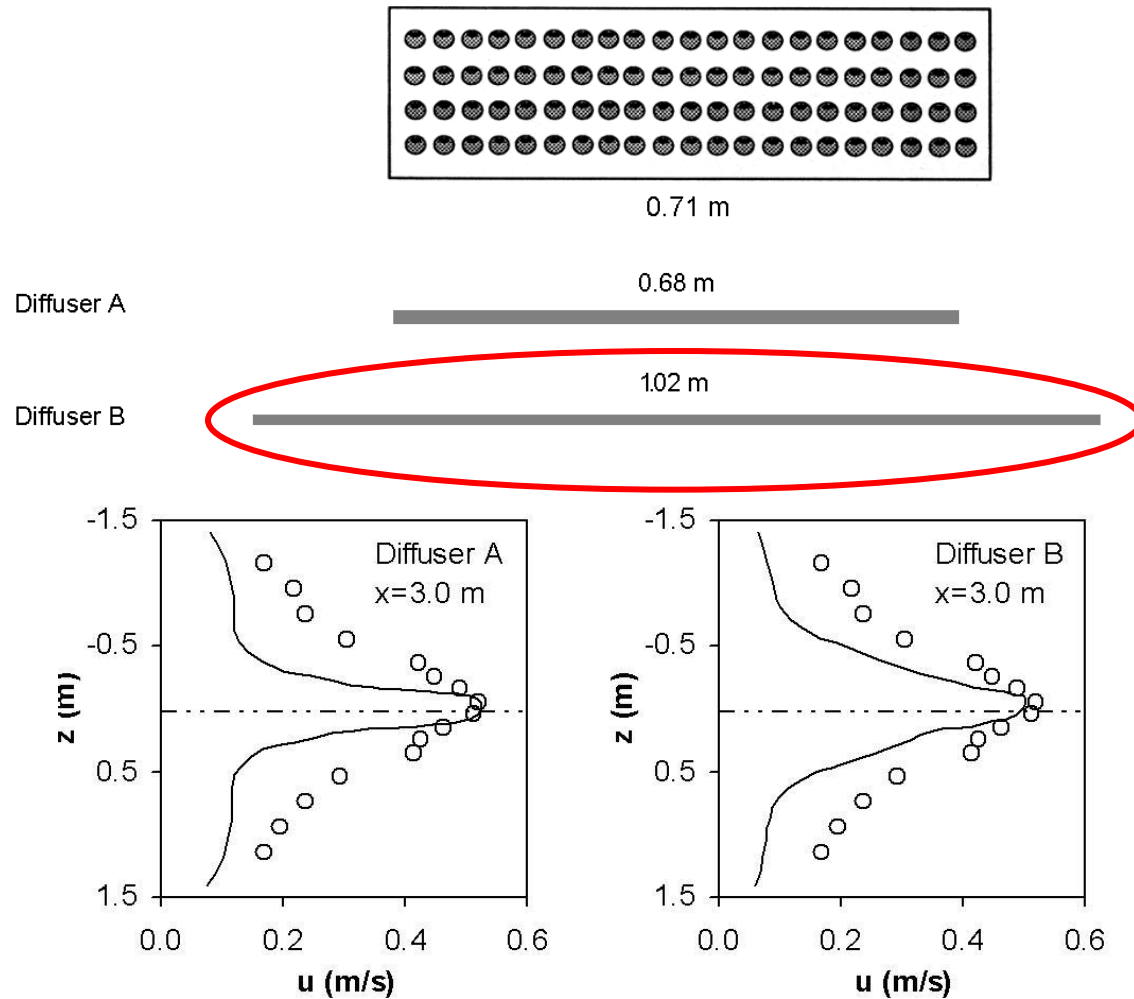
The test room is the IEA Annex 20 room with length, width and height equal to 4.2 m, 3.6 m and 2.5 m.

The heat load consists of two PCs, two desk lamps and two manikins producing a total heat load of 480 W. One work place is used in some of the experiments (240 W).

# Case Study, Mixing Ventilation with Wall Mounted ATD

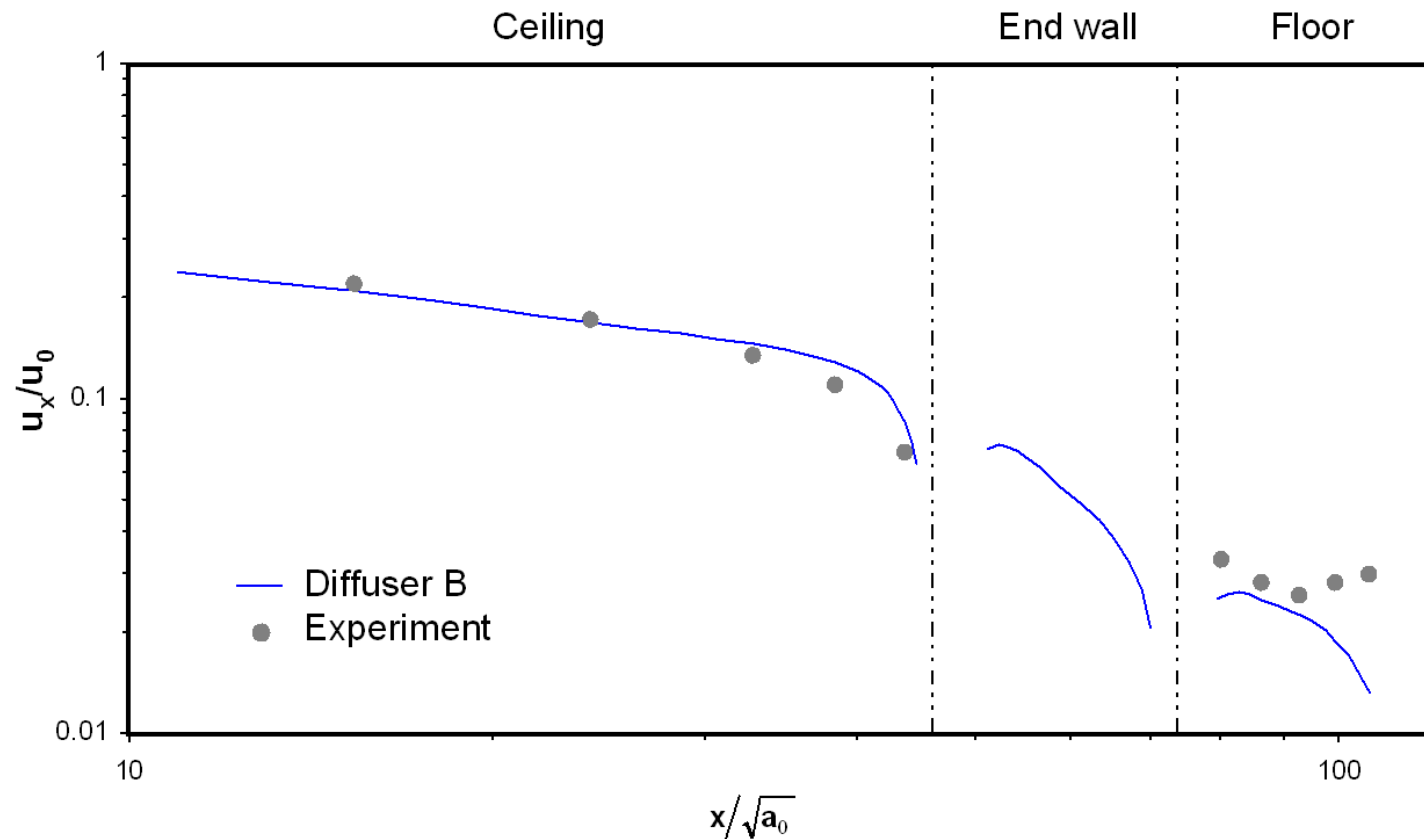


# Simulation of the Diffuser

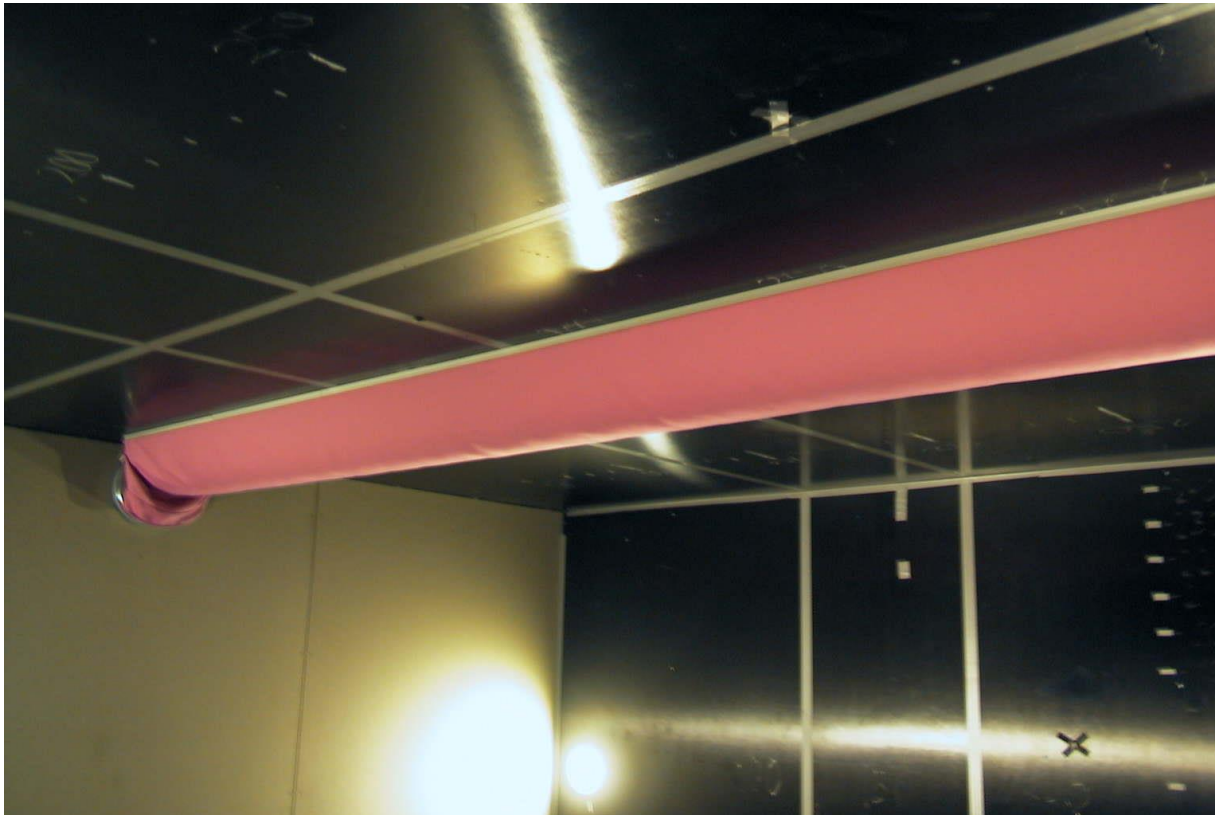




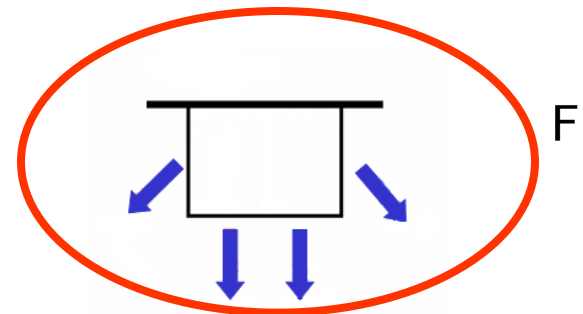
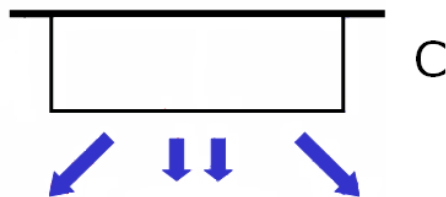
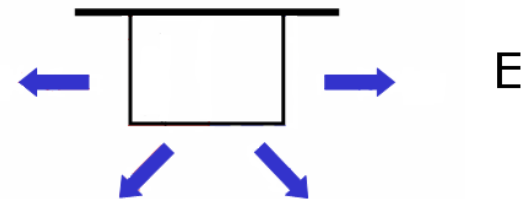
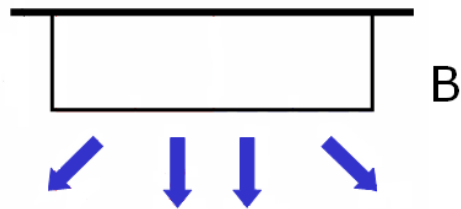
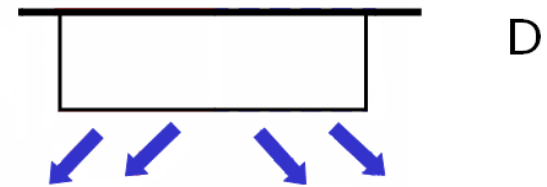
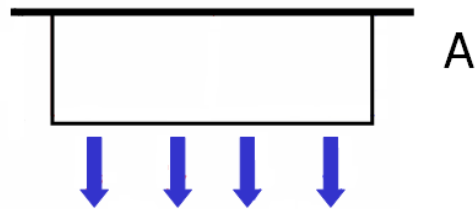
# Mixing Ventilation with End Wall Mounted Diffuser



# Case Study, Vertical Ventilation

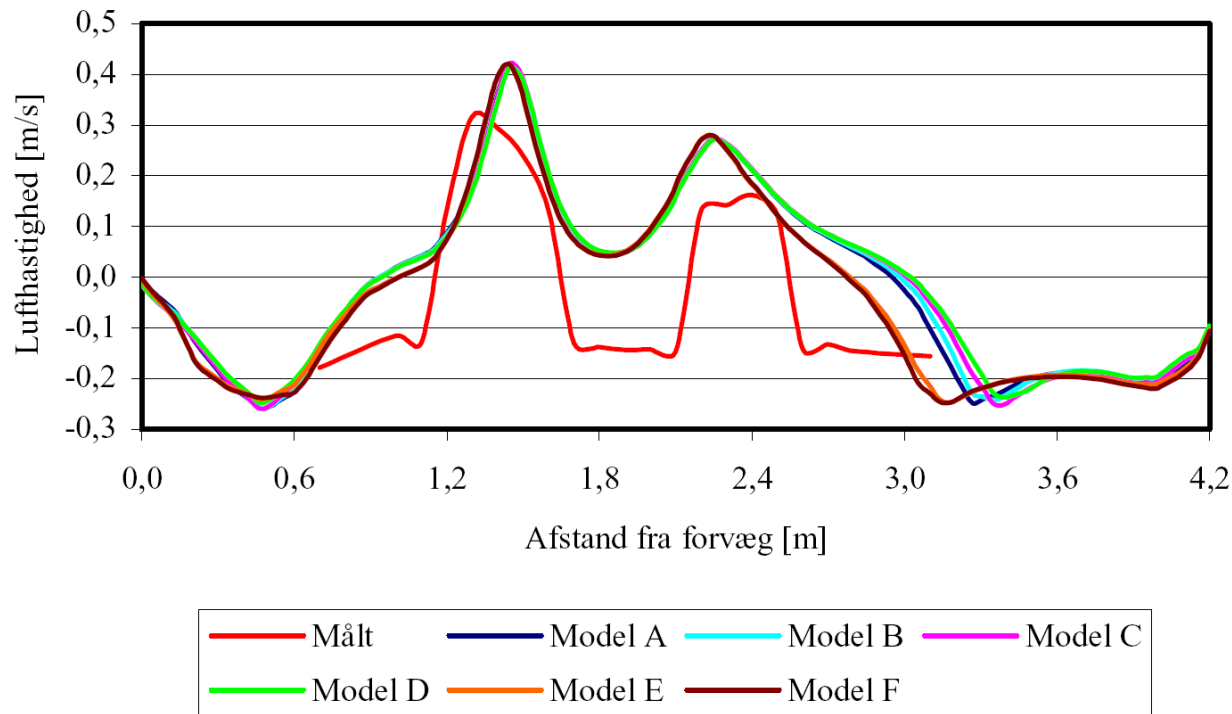


# Vertical Ventilation, Diffuser

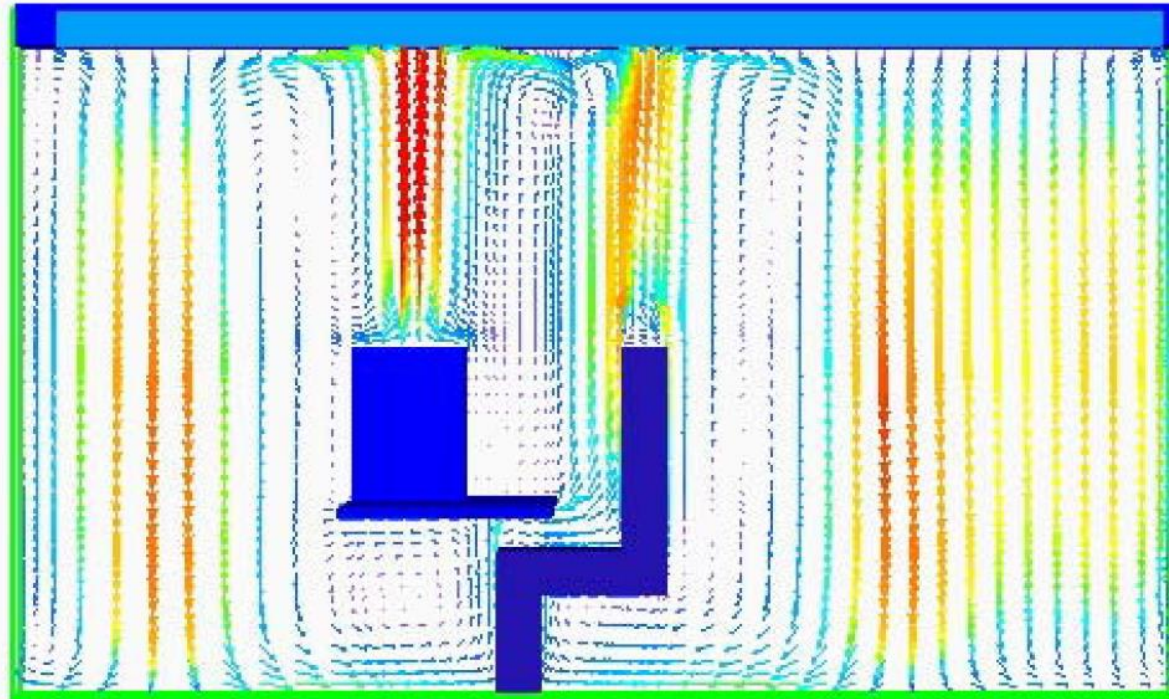


# Vertical Ventilation, Diffuser

1st order steady state equations,  $k$ - $\epsilon$  turbulence model, 300,000 cells



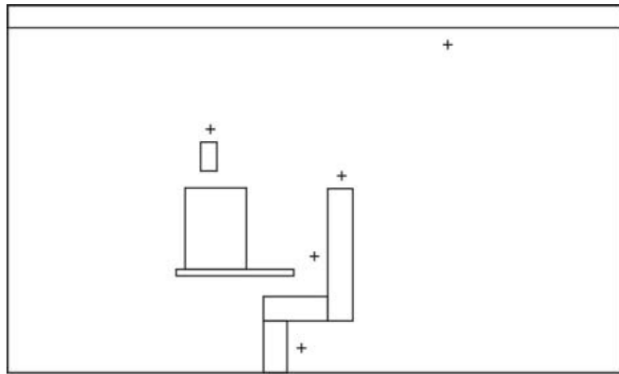
# Vertical Ventilation



BC: Diffuser F

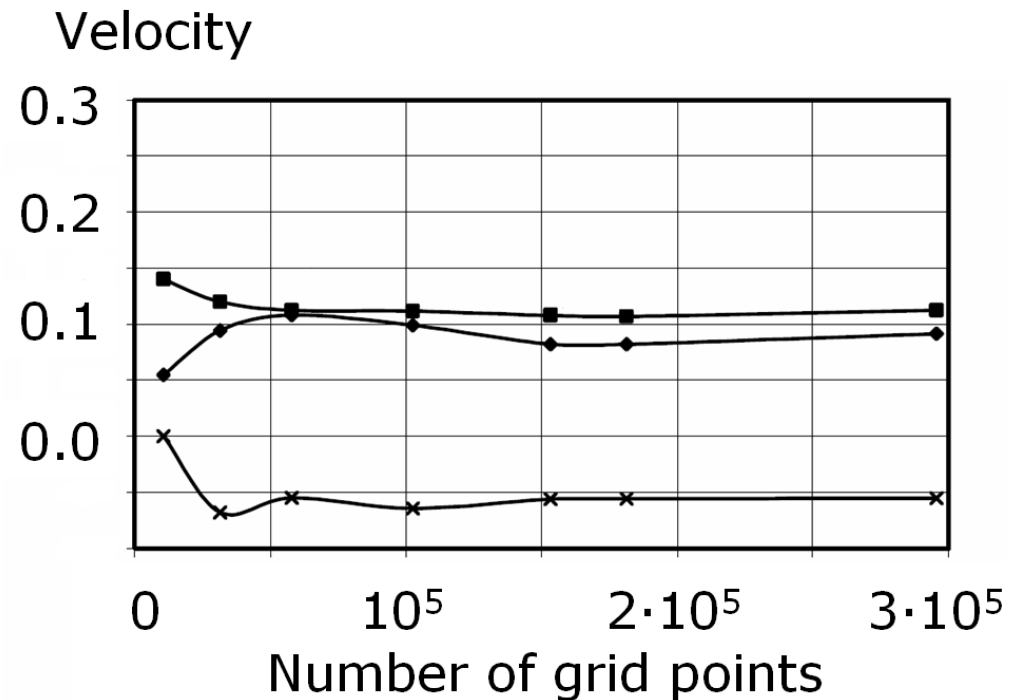


# Vertical Ventilation, Quality Control



Velocity in  
y-direction

Monitoring points

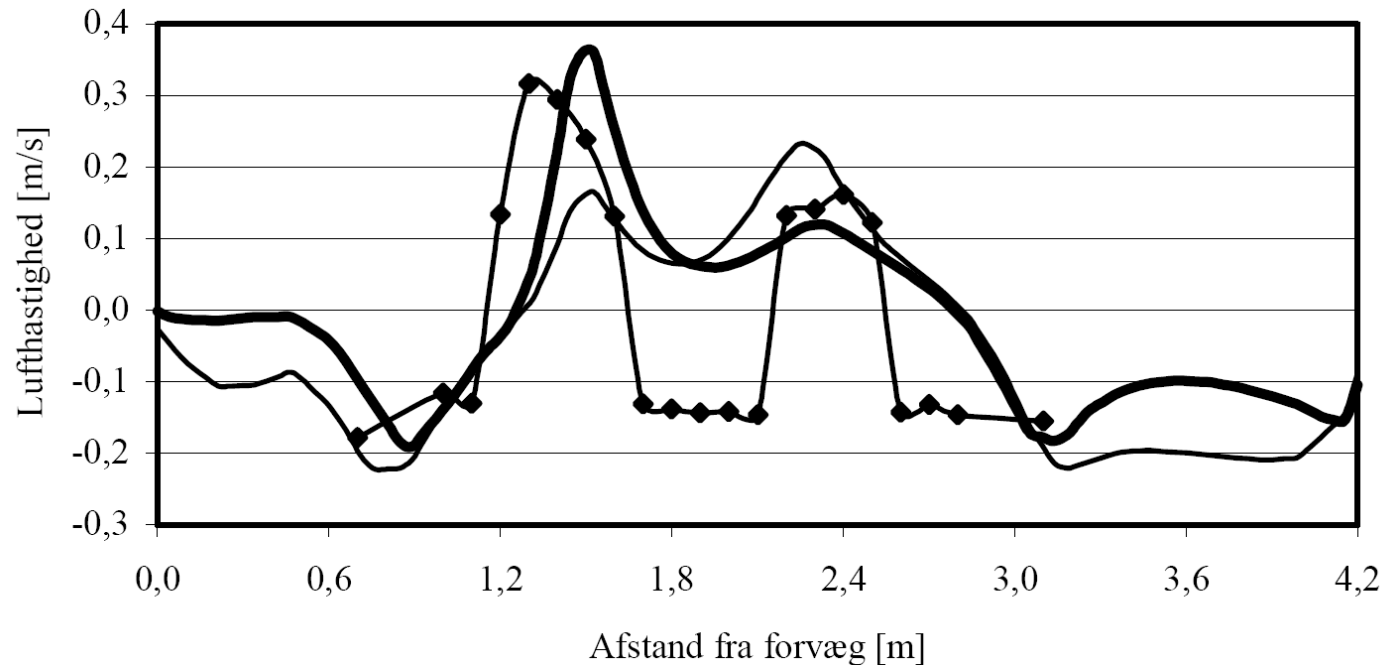
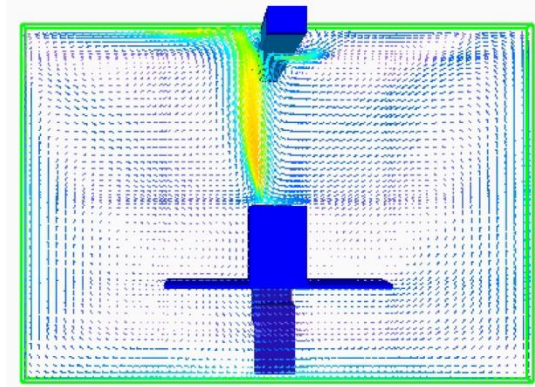


# Vertical Ventilation

Predictions in the **whole**  
room

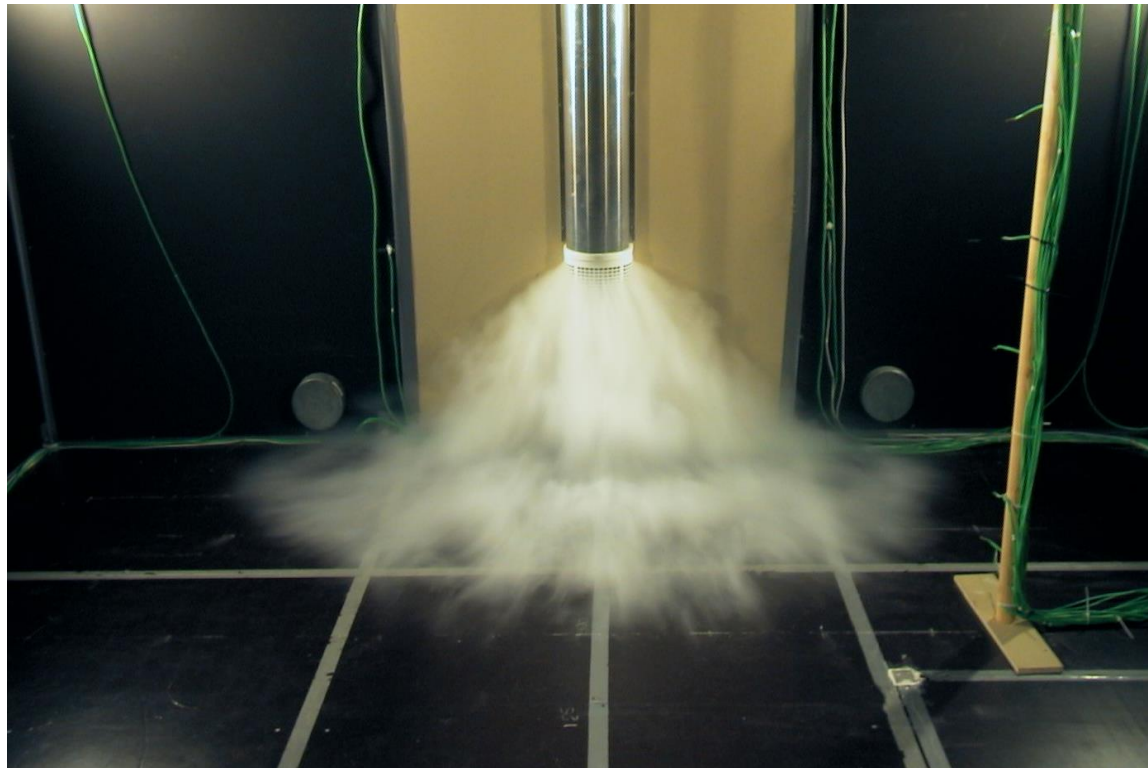
$$n = 5 \text{ h}^{-1}$$

218.400 grid points



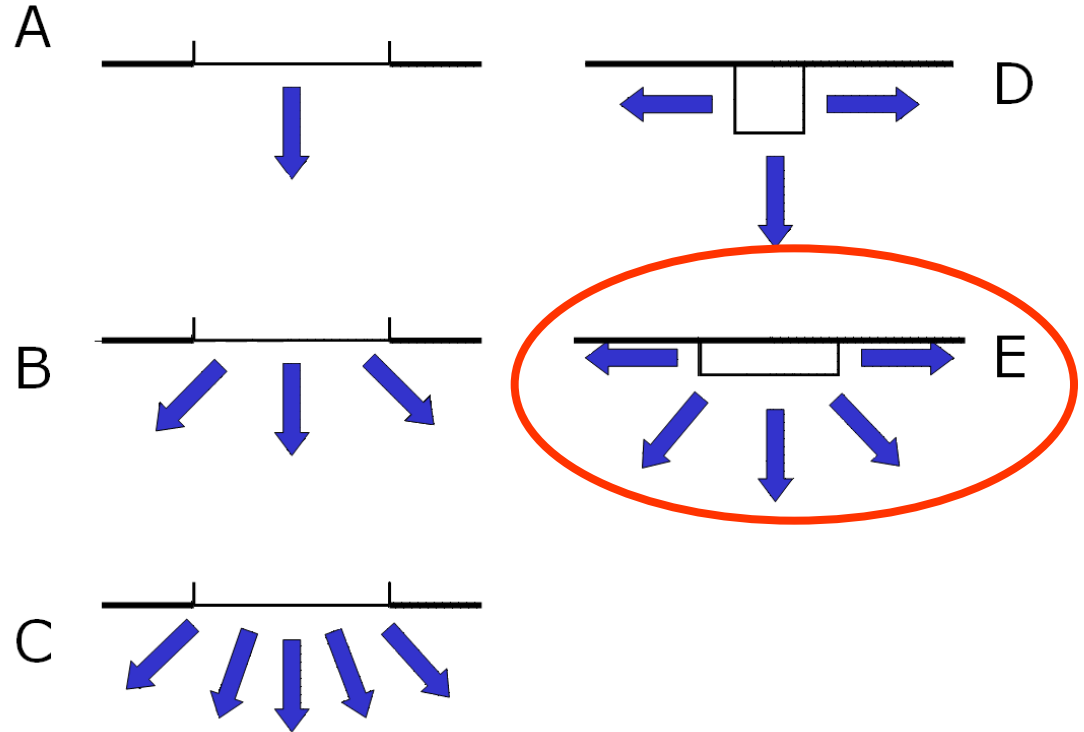
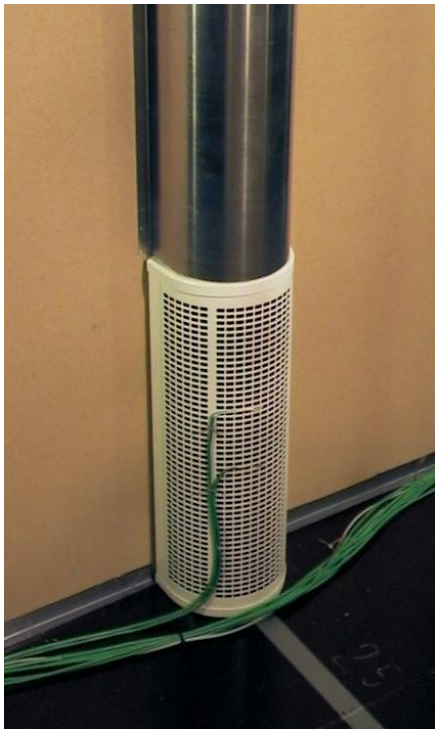


# Case Study, Displacement Ventilation

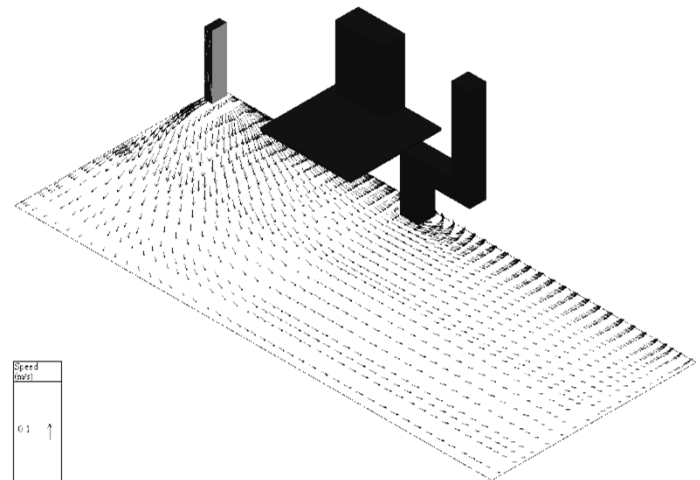
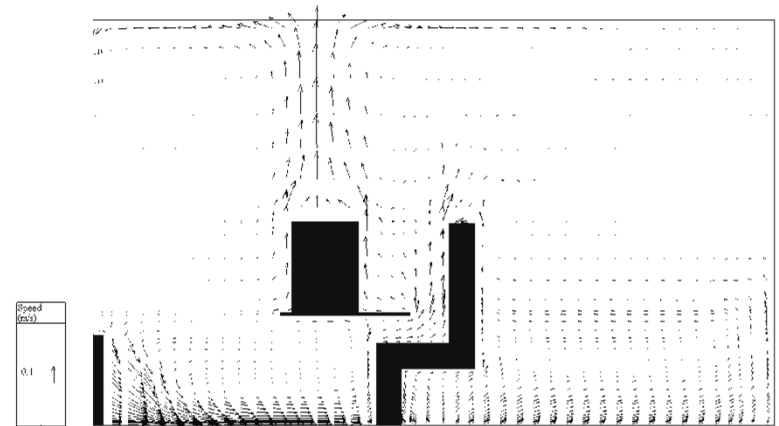
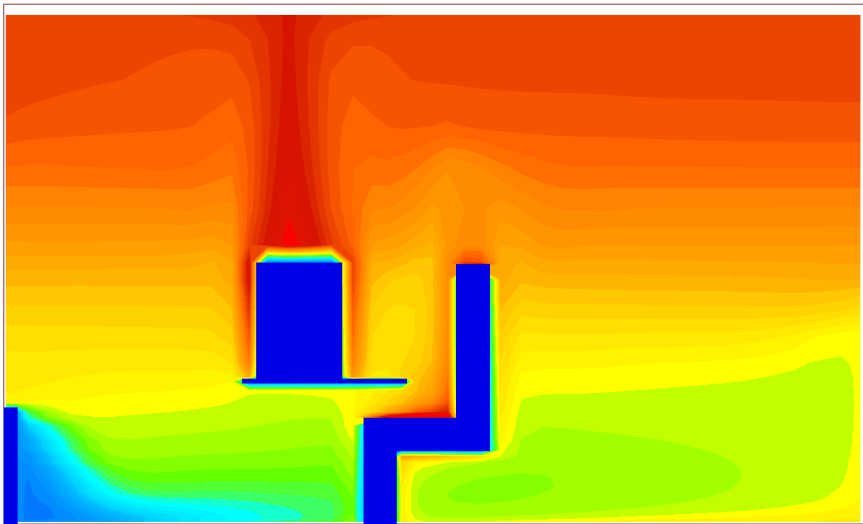




# Diffuser for Displacement Ventilation



# Displacement Ventilation



# Case Study, Mixing Ventilation with Ceiling Mounted ATD

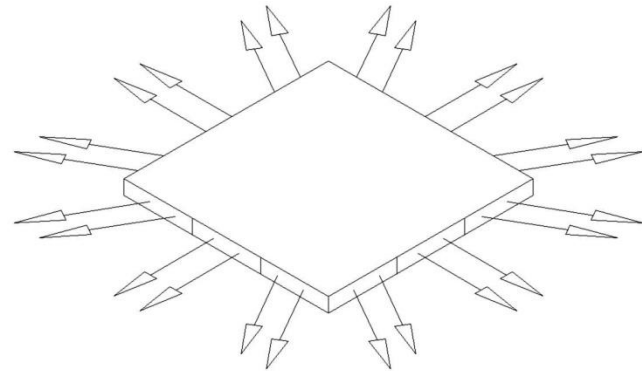


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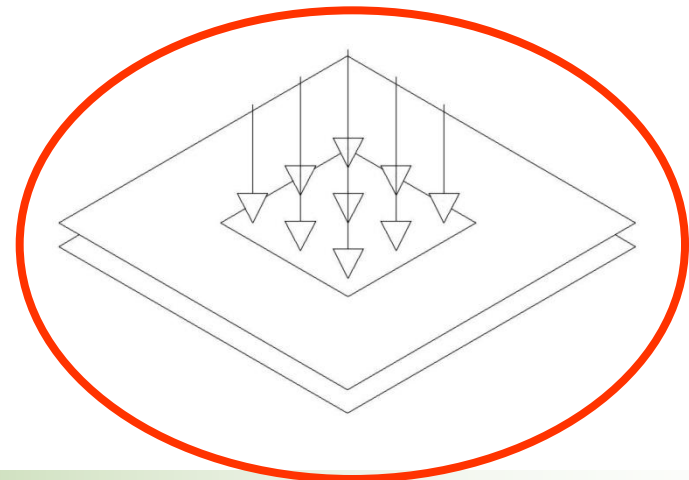


Federation of European Heating, Ventilation and Air-conditioning Associations

# Mixing Ventilation with Ceiling Diffuser, Diffuser Models

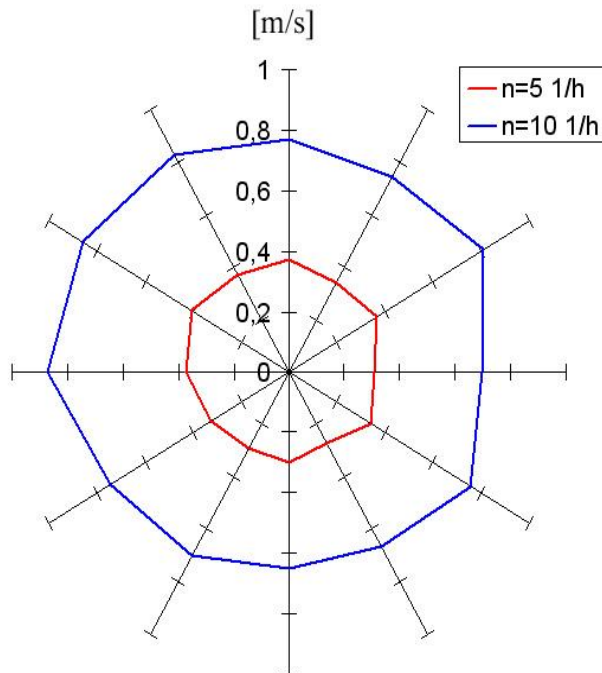


- Fixed-flow diffuser
- Diffuser with horizontal surface

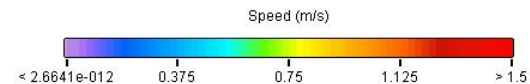
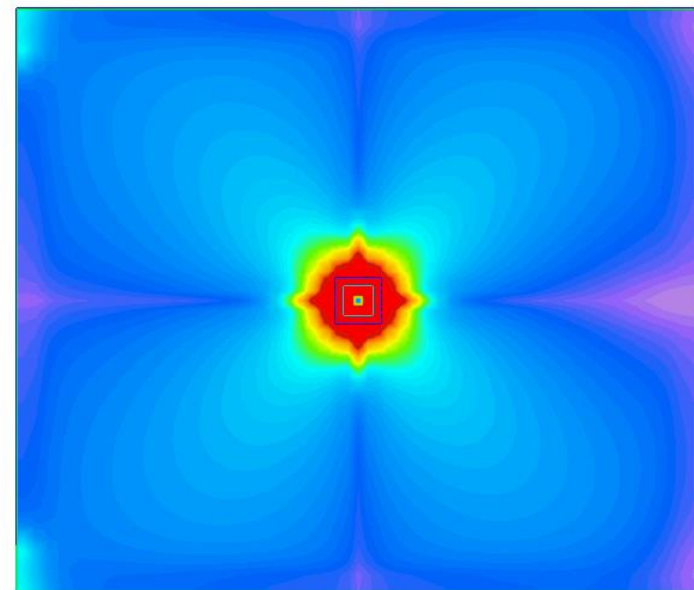


# Diffuser Model

## Measurements

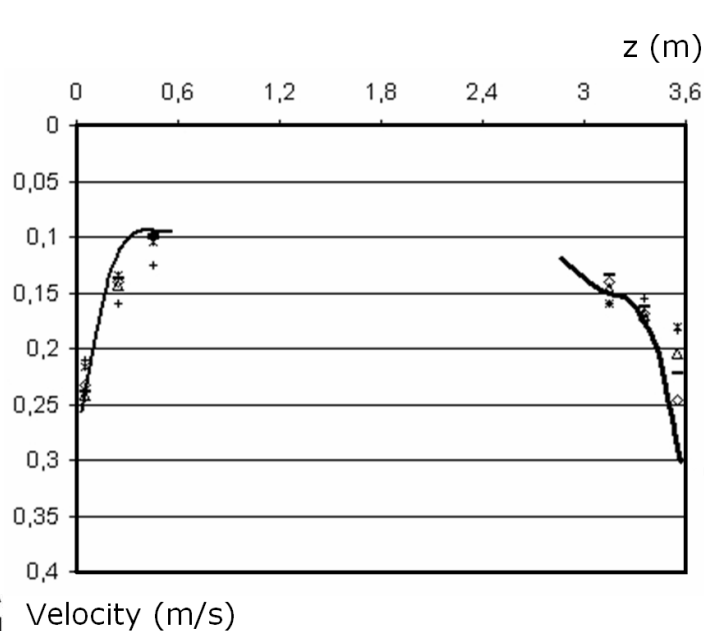
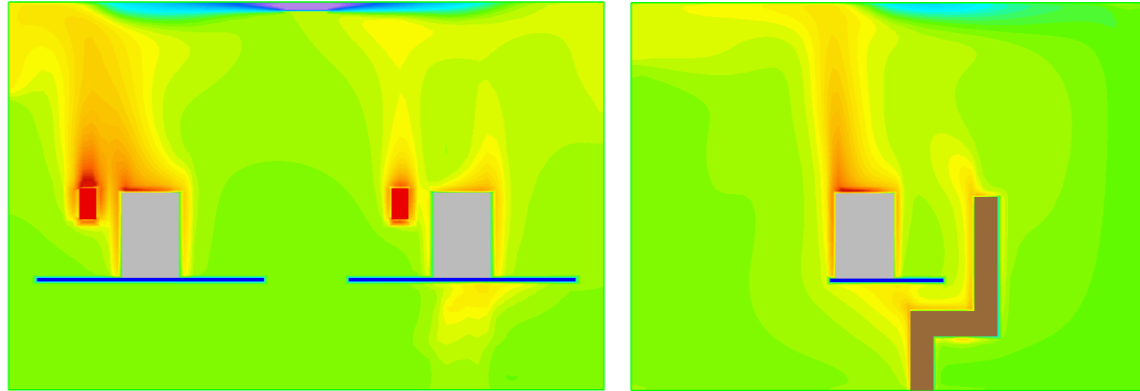


## Diffuser with horizontal surface



# CFD Simulations

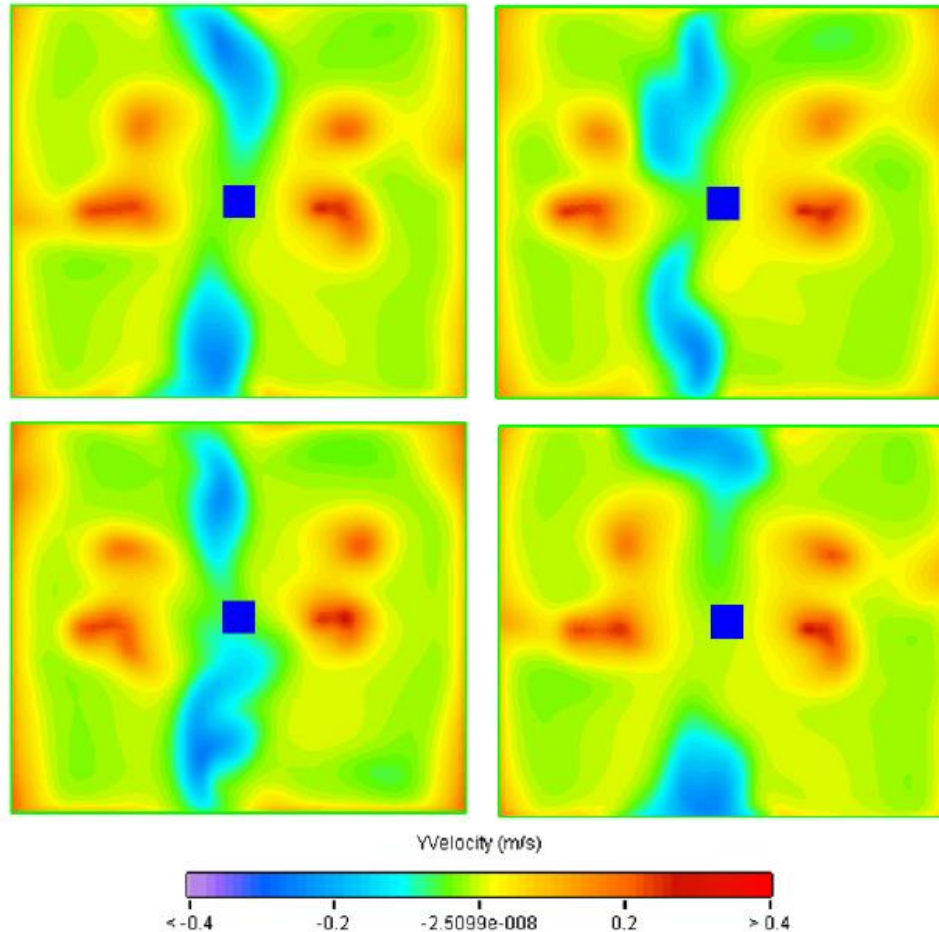
Temperatures



Velocities at the the height of 1.80 m close to the side walls

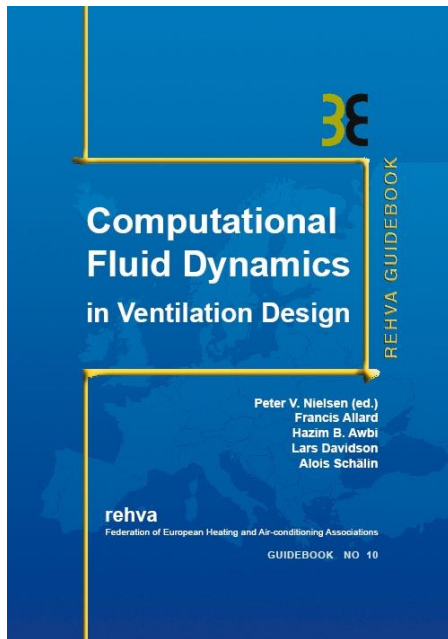
# Simulation of Unsteady Flow

(Time Dependent Equations)





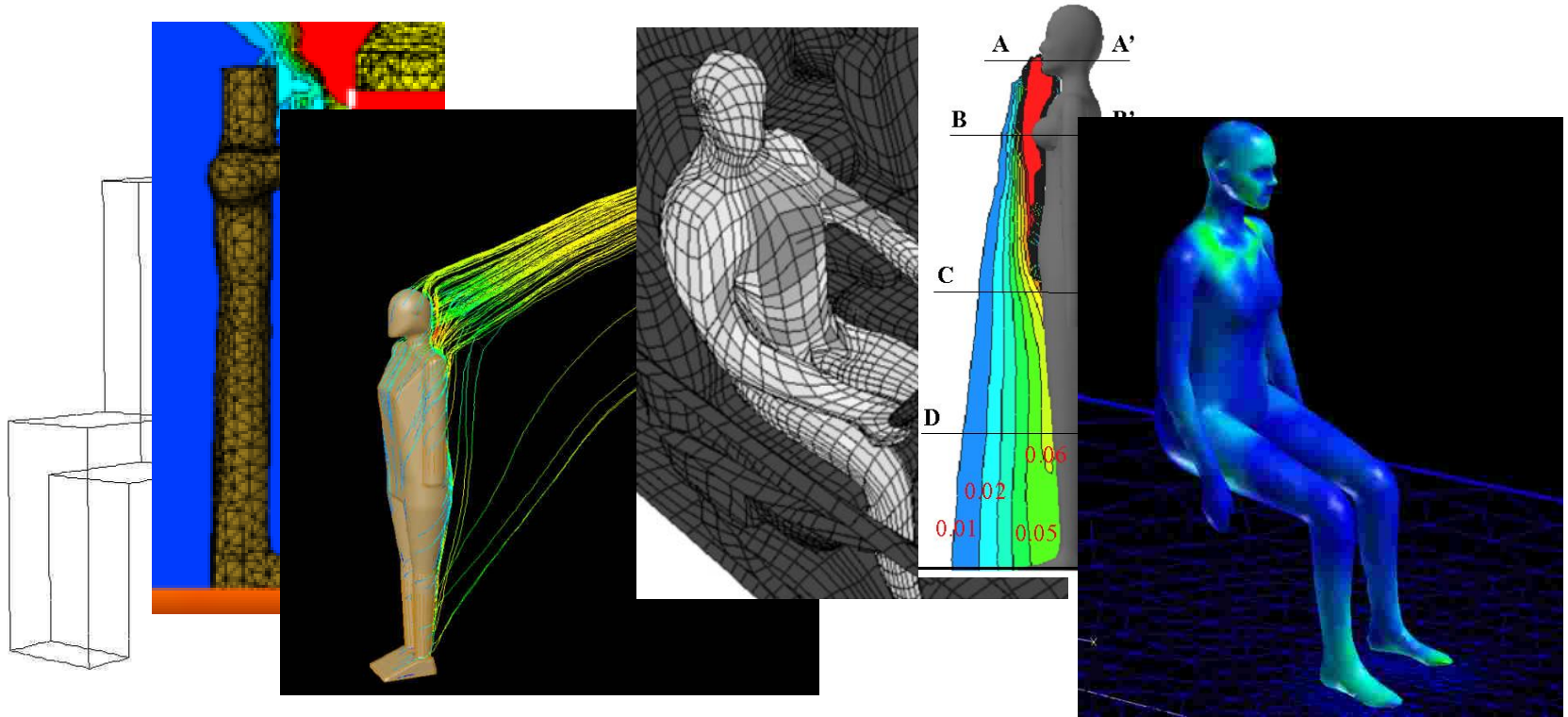
# CFD in Ventilation Design



Computational fluid dynamics in a nutshell  
Symbols and glossary  
Mathematical background  
Turbulence models  
Numerical methods  
Boundary conditions  
Quality control  
CFD combined with other prediction models  
Application of CFD codes in building design  
Case studies  
**Benchmark tests**



# CFD Manikins



Benchmark test for different CFD manikins:  
[www.cfd-benchmarks.com](http://www.cfd-benchmarks.com)

# Additional literature

P. V. Nielsen, The Selection of Turbulence Models for Prediction of Room Airflow. ASHRAE Transactions. 1998; Vol. 104, Part 1B. pp. 1119-1127. ISSN 0001-2505.

D. N. Sørensen and P. V. Nielsen, Quality Control of Computational Fluid Dynamics in Indoor Environments. International Journal of Indoor Environment and Health, Vol. 13, no. 1, pp. 2-17, March 2003.

D. N. Sørensen and P. V. Nielsen, Guest editorial: CFD in Indoor Air. International Journal of Indoor Environment and Health, Vol. 13, no. 1, p. 1, March 2003.

P. V. Nielsen, Computational Fluid Dynamics and Room Air Movement. Indoor Air, International Journal of Indoor Environment and Health, Vol. 14, Supplement 7, pp. 134 – 143, 2004.

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